

# Design of a Single-Point Cutting Tool

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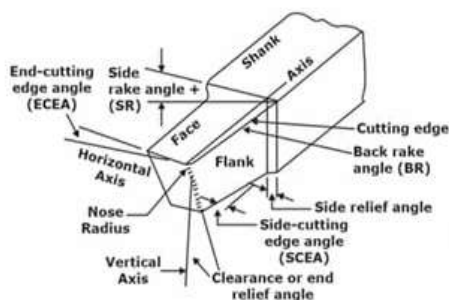
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**ABSTRACT** –A single-point cutting tool is the type of cutting tool or cutter that removes material by means of one cutting edge during a single stroke of movement. This paper deals with the design of a single-point cutting tool for machining medium carbon steel with a cutting velocity of 400 m/min, feed of 0.5mmperrevolutionanddepthofcutof0.5mm. Single point cutting tool is used for important operation in the lathe working facing and turning hence study of geometry of a single point cutting tool is very important part of study required for manufacturing. The single point cutting tool design involves collecting data from the data book. It is an iterative process that involves varying the angle within a certain limit provided by the data to get an optimum result.

**KeyWords:** Single point cutting tool, design, medium carbon steel, carbide tool, power consumed, lathe, American standard association, orthogonal rake system, vector method of conversion.

## I. INTRODUCTION

Single point cutting tool is a tool which touches the work piece at only one point. An operation like turning, facing, shaping, planing, boring etc use single-point cutting tool. Shank. As an important part of tool increasing the durability and life of the tool material is an important factor while making a tool. A small change of even 1 in the tool parameters can alter the force encountered by the tool by more than ten times. Hence the proper design of the tool is very important .



Single point cutting tool

In this paper, we designed a single point cutting tool based on a proper condition that can be extended to any variation of data. In this paper we have taken the following prerequisite-

- Work piece material – cast iron or mild steel
- Cutting velocity [Vc] - 36.576meter/minute
- Feed[S<sub>0</sub>]-:0.5mm/rev
- Depth of cut[t]-:0.5mm
- 

The important properties of the work material are like as

- Ultimate Tensile Strength– 827 MPa
- Tensile Strength Yield - 124Mpa
- Modulus of Elasticity - 200GPa
- Density -7.8\*10<sup>3</sup>kg/m<sup>3</sup>
- Brinell Hardness Number [BHN] -110
- Shear Modulus - 77.5GPa

Before designing cutting tool we know of the requirements. The requirements of the cutting tool are as

- 1)The cutting tool and the work material must be held rigidly so the tool can penetrate the work piece when forces are applied.
- 2)The shank of the tool must be properly analyzed for strength and rigidity.
- 3)The deflection of the tool must be within a certain limit.

## II. MATERIAL SELECTION

The required properties of tool material are listed below

- The tool material is harder than the work piece.
- The tool material must be chemically inert.
- The material must be able to resist wear and tear.
- The material should be thermally stable.
- The material should have good thermal conductivity and less coefficient of thermal expansion.

Some common Tool materials most widely used are:-

- Carbide
- Ceramic
- Cubic Boron Nitride
- Diamond
- High speeded steel
- Tool with coating of Titanium Carbide(TiC) and Titanium Nitride (TiN).

In our design we electe Carbide tool with a coating of TiC as BHN of Carbide is much higher than that of medium carbon steel.

### III. DETERMINATION OF ANGLES

From the data book , the recommended angles for carbide tool in ASA system.

- Back rake angle( $Y_y$ ) -0
- Side rake angle( $Y_x$ ) -6
- End clearance angle( $\alpha_y$ ) -5
- Side clearance angle( $\alpha_x$ ) -5
- Side cutting edge angle( $\phi_s$ )-15
- Approach angle( $\phi$ ):- 75
- Nose Radius( $r$ ) -1 mm

#### 3.1 Conversion from ASA system ORS

We have used the vector method of conversion from ASA system to ORS

$$\begin{aligned} 1] \tan(Y_0) &= \tan(Y_x) \sin(\phi) + \tan(Y_y) \cos(\phi) \\ &= \tan(Y_0) = \tan(6) \sin(75) + \tan(0) \cos(75) \\ &= \tan(Y_0) = 0.1015 \\ &= Y_0 = \tan^{-1}(0.1015) \\ &Y_0 = 5.795 \end{aligned}$$

$$\begin{aligned} 2] \tan(\lambda) &= \tan(Y_y) \sin(\phi) - \tan(Y_x) \cos(\phi) \\ &= \tan(\lambda) = \tan(0) \sin(75) - \tan(6) \cos(75) \\ &= \tan(\lambda) = -0.0272 \\ &= \lambda = \tan^{-1}(-0.0272) \\ &= \lambda = -1.568 \end{aligned}$$

$$\begin{aligned} 3] \cot(\alpha_0) &= \cot(\alpha_x) \sin(\phi) + \cot(\alpha_y) \cos(\phi) \\ &= \cot(\alpha_0) = \cot(5) \sin(75) + \cot(5) \cos(75) \\ &= \cot(\alpha_0) = 13.998 \\ &= \tan(\alpha_0) = 0.0714 \\ &= \alpha_0 = \tan^{-1}(0.0714) \\ &= \alpha_0 = 4.083 \end{aligned}$$

Hence the angles in ORS (Orthogonal Rake System) are as

- $\lambda$  -In clination Angle--1.558
- $Y_0$  -Orthogonal Rake Angle-5.795
- $\alpha_0$  -Orthogonal Clearance Angle-4.083
- $\alpha_0'$  -Auxiliary Clearance Angle-2
- $\phi_e$  -Auxiliary Cutting Edge Angle-15
- $\phi$  -End Cutting Edge Angle-75
- $r$  -Nose Radius-1mm

### CALCULATION OF DYNAMIC SHEAR STRESS

Dynamic shear stress is the product of Brinell hardness number and 0.186.

$$\begin{aligned} \text{We know, } \tau &= 0.186 * \text{BHN} \\ &= 0.186 * 110 \\ &= 20.46 \text{ kg/mm}^2 \\ &= 204.6 \text{ N/mm}^2 \end{aligned}$$

### DETERMINATION OF CHIP REDUCTION COEFFICIENT( $\zeta$ )

Chip Reduction Coefficient is defined as the ratio of chip thickness before cut to the chip thickness after cut .It indicates the degree of deformation.

Assuming it to be rough turning cycle the chip reduction coefficient is assumed to be 2.5 and Factor of safety [FOS] is assumed as 10.

### CALCULATION OF TANGENTIAL FORCE

Tangential force is defined as the component of force which acts on the edge of the cutting tool along at tangent to cutting tool body.

$$\begin{aligned} P_Z &= t s_0 \tau_s (\zeta - \tan(Y_0) + 1) \text{ Here,} \\ t &\text{ is the thickness of the work piece} \\ s_0 &\text{ is the feed,} \\ P_Z &= 0.5 * 0.5 * 303.18 (2.5 - 0.1015 + 1) \\ &= 173.83 \text{ N} \end{aligned}$$

### CALCULATION OF LONGITUDINAL FEED FORCE

$$\begin{aligned} P_x &= t s_0 \tau_s (\zeta - \tan(Y_0) - 1) \sin(\phi) \text{ Here,} \\ t &\text{ is the thickness of the work pieces } s_0 \text{ is the feed} \\ &= 0.5 * 0.5 * 204.6 (2.5 - 0.1015 - 1) \sin(75) \\ &= 69.07 \text{ N} \end{aligned}$$

### CALCULATION OF RADIAL FORCE

$$\begin{aligned} P_y &= t s_0 \tau_s (\zeta - \tan(Y_0) - 1) \cos(\phi) \\ &= 0.5 * 0.5 * 204.6 (2.5 - 0.1015 - 1) \cos(75) \\ &= 18.50 \text{ N} \end{aligned}$$

### CALCULATION OF CROSS-SECTIONAL DIMENSION

We assume that

H/B must be between 1.25-1.6, Here

H is the height the tool

B is the breadth of the tool

The effective length  $L_e$  of the tool must be between 25-30mm for proper holding of the tool.

So, we assume H/B = 1.5 and  $L_e = 30$  mm We Know,

Total stress is given by the sum of principal stresses i.e.,  $\sigma = \sigma_1 + \sigma_2$

$$\sigma = \frac{6P_z L_e}{BH^2} + \frac{6P_x L_e}{HB^2} \dots \dots \dots (i) \dots (1)$$

Now,

$$\sigma = \frac{\sigma_{ut}}{FOS} = \frac{565MPa}{10} = 56.5 MPa \quad (2)$$

Now, from(1)and(2),

Sigma =82.7Mpa

$$82.7 = \frac{6 \cdot 257.596 \cdot 30}{2.25 \cdot B^3} + \frac{6 \cdot 102.39 \cdot 30}{1.5 \cdot B^3}$$

B=15.46 mm

Assuming a nearest standard value of the integer we get B=15mm

From the recommended set of cross section 15\*10 is available.

So the Cross-Section of the cutting Tool is 15\*10mm.

#### POWER CONSUMED

The power consumed in the cutting operation the work piece is given by the equation

P= Pz\*Vc Here,

Pz is the Tangential Force Vc is the cutting velocity

={(173.83\*36.57)/60} Watt

=105.94Watt

=0.105 KW

#### DEFLECTION

$$\delta = \frac{4PzLe^3}{EBH^3}$$

= 0.00625788 mm

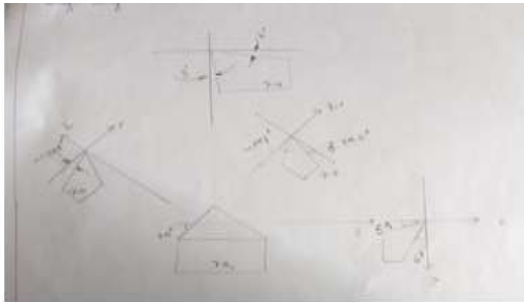
=6.25788microns

#### TOOL SIGNATURE

The expected tool signature in ORS System is

-1.558-5.795-4.083-2-15-75-1

#### ROUGHDIAGRAM



#### IV. CONCLUSION

This work deals with the design of single point cutting tool and after the determination of

forces the effect of these forces were studied. With all these forces it can be summarized that the design of the single point cutting tool is strong enough to be ar cutting for ces during all kind of lathe operation.

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